Nonlinear systems Course introduction

### G. Ferrari Trecate

Dipartimento di Ingegneria Industriale e dell'Informazione Università degli Studi di Pavia

Advanced automation and control

### Course schedule

### Advanced automation and control

Industrial automation + Nonlinear systems

### Lectures

- Monday 14-16 room EF3, Thursday 16-18, room E1 (Industrial automation)
- Wednesday 14-16, room E1 (Nonlinear systems)

### Office hours

• By appointment (giancarlo.ferrari@unipv.it). Office: Dipartimento di Ingegneria Industriale e dell'Informazione, floor F

### Course website

### http:

//sisdin.unipv.it/labsisdin/teaching/courses/ails/files/ails.php

 a copy of the slides can be downloaded after authentication with login/password

### Textbooks

For a review of basic systems theory and automatic control

- G. F. Franklin, J. D. Powell, A. Emami-Naeini. *Feedback Control of Dynamic Systems* 6th ed., 2009 Prentice Hall
- P. Bolzern, R. Scattolini, N. Schiavoni. Fondamenti di Controlli Automatici, 2nd ed., 2004, McGraw-Hill, Italia

For the topics in nonlinear systems covered in the course

- J.-J. E. Slotine e W. Li. Applied nonlinear control. Prentice-Hall (1991)
- H.K. Khalil. Nonlinear systems third edition. Prentice-Hall (2002)
- S. Sastry. Nonlinear systems Analysis, Stability and Control. Springer-Verlag (1999) (and C. Tomlin - slides of the course "Advanced Nonlinear Control", Stanford University)
- All above books cover several topics that will be not discussed in the course. Khalil an Sastry's books are the most advanced (and difficult) ones
- The exam will focus only on topics covered in the course

Ferrari Trecate (DIS)

Nonlinear systems

### Exams

Closed-books closed-notes written exam split in two parts

- First part: industrial automation
- Second part: nonlinear systems

Total duration:  $\sim$  3h. No graphic or programmable calculators are allowed

### Registration to exams

Through the university website

Usually, registrations end 7 days before the exam date

### Nonlinear (NL) systems

### Analysis vs. simulation

- Steadily increasing computing power allows one to simulate complex NL systems
- Simulation and intuition allow one to understand several aspects of NL systems

### However,

- Impossible to use only simulation to prove interesting properties (e.g. stability)
- Analysis procedures allow properties of NL systems to be rigorously assessed
  - Sometimes, results are surprising and highlight behaviors one had not tought to simulate !

### Nonlinear (NL) systems

### NL systems vs. linear systems

Several results on the analysis and control of linear systems HOWEVER

- Most real systems are NL
- Linear systems do not capture behaviors such as
  - isolated multiple equilibria
  - limit cycles
  - subharmonics
  - complex dynamics, e.g. chaos

### Next ...

- Review of systems theory !
- Examples of nonlinear behaviors

### Review

### NL system

$$\dot{x}(t) = f(x(t), u(t), t)$$
 (1)  $x(t) \in \mathbb{R}^n$  state  
 $y(t) = g(x(t), u(t), t)$  (2)  $u(t) \in \mathbb{R}^m$  input  
 $x(t_0) = x_0$  (3)  $y(t) \in \mathbb{R}^p$  output

- (1): state equation
- (2): output equation
- n: system order

### Definition

A state trajectory is a function x(t) verifying (1) and (3). For highlighting the dependence on the input, initial time and initial states, we write  $x(t) = \phi(t, t_0, x_0, u)$  and  $\phi$  is called *transition map* 

### Drawing state trajectories

Often one draws the image of the trajectory  $\phi(t, t_0, x_0, u)$ , i.e. the set of points

 $\{\phi(t, t_0, x_0, u), t \geq t_0\} \subset \mathbb{R}^n$ 



### Review

### NL system

 $\dot{x}(t) = f(x(t), u(t), t)$ y(t) = g(x(t), u(t), t) $x(t_0) = x_0$ 

 $egin{aligned} x(t) \in \mathbb{R}^n \ u(t) \in \mathbb{R}^m \ y(t) \in \mathbb{R}^p \end{aligned}$ 

An NL system is:

- Invariant if f and g do not depend upon time
  - ▶ Without loss of generality, one can set  $t_0 = 0$  and  $\phi(t, t_0, x_0, u) = \phi(t, x_0, u)$
- Autonomous if the system does not depend upon the input u(t)

$$\phi(t, t_0, x_0, \boldsymbol{u}) = \phi(t, t_0, x_0)$$

• Invariant and autonomous:  $\dot{x}(t) = f(x(t)), \ y(t) = g(x(t))$ 

$$\phi(t, t_0, x_0, u) = \phi(t, x_0)$$

- Static if n = 0
  - described only through the output equation y(t) = g(u(t), t)

### Review of linear systems

A system is linear if f and g are linear functions of x and u

 $\dot{x}(t) = A(t)x(t) + B(t)u(t)$ y(t) = C(t)x(t) + D(t)u(t)

A(t), B(t), C(t), D(t) matrices

Linear Time-Invariant (LTI) system

 $\dot{x}(t) = Ax(t) + Bu(t)$ y(t) = Cx(t) + Du(t)

A, B, C, D matrices

## Multiple isolated equilibria

### NL vs. linear systems: Duffing oscillator



### Model

$$ml^2\ddot{x}_1 = mgl\sin(x_1) - \alpha x_1 - k\dot{x}_1 + \tau$$

• 
$$-\alpha x_1$$
: restoring torque ( $\alpha > 0$ )

• 
$$-k\dot{x}_1$$
: damping torque  $(k > 0)$ 

•  $\tau$ : electromagnetic torque (input)

### NL system

Defining 
$$x_2 = \dot{x}_1$$
,  $u = \frac{\tau}{ml^2}$ ,  
 $\dot{x}_1 = x_2$   
 $\dot{x}_2 = \frac{g}{l} \sin(x_1) - \frac{\alpha}{ml^2} x_1 - \frac{k}{ml^2} x_2 + u$ 

### Review

### Equilibrium

Given  $u(t) = \bar{u}$ ,  $\forall t \ge 0$ , the state  $\bar{x} \in \mathbb{R}^n$  is an equilibrium state for the nonlinear time-invariant system  $\dot{x} = f(x, u)$  if it verifies  $f(\bar{x}, \bar{u}) = 0^a$ . The pair  $(\bar{x}, \bar{u})$  is called an equilibrium.

 $x^{*}\dot{x} = x^{2} + 1$ ,  $x(t) \in \mathbb{R}$  has no equilibrium state

# Duffing oscillator: equilibra for $\bar{u} = 0$ Physical intuition: 3 equilibra

### Duffing oscillator: equilibria of approximate models

### NL system

$$\dot{x}_1 = x_2$$
  
 $\dot{x}_2 = \frac{g}{l}\sin(x_1) - \frac{\alpha}{ml^2}x_1 - \frac{k}{ml^2}x_2 + u$ 

Linear approximation: 
$$sin(x_1) \simeq x_1$$
  
LTI system  $(u = 0)$   
Equilibrium states:  
 $\dot{x}_1 = x_2$   
 $\dot{x}_2 = \left(\frac{g}{l} - \frac{\alpha}{ml^2}\right) x_1 - \frac{k}{ml^2} x_2$   
 $\left(\frac{g}{l} - \frac{\alpha}{ml^2}\right) \neq 0 \Rightarrow \bar{x}_1 = 0$ 

Either one or infinite equilibrium states

### Duffing oscillator: equilibria of approximate models

NL system

$$\dot{x}_1 = x_2$$
  
 $\dot{x}_2 = \frac{g}{l} \sin(x_1) - \frac{\alpha}{ml^2} x_1 - \frac{k}{ml^2} x_2 + u$ 

Approximation: 
$$sin(x_1) \simeq x_1 - x_1^3/6$$

Approximated NL system (u = 0) Equilibrium states:

$$\dot{x}_1 = x_2 \qquad \qquad \bar{x}_2 = 0$$
$$\dot{x}_2 = \left(\frac{g}{l} - \frac{\alpha}{ml^2}\right) x_1 - \frac{g}{6l} x_1^3 - \frac{k}{ml^2} x_2 \qquad \left(\frac{g}{l} - \frac{\alpha}{ml^2}\right) \bar{x}_1 - \frac{g}{6l} \bar{x}_1^3 = 0$$

One can have 3 equilibrium states

### Duffing oscillator: equilibria of the approximate NL model

If we set 
$$rac{g}{l}-rac{lpha}{ml^2}=1$$
,  $rac{g}{6l}=1$ ,  $rac{k}{ml^2}=\eta$ , we get

Automonous Duffing model:

Equilibrium states:

$$\begin{aligned} \dot{x}_1 &= x_2 \\ \dot{x}_2 &= x_1 - x_1^3 - \eta x_2 \end{aligned} \qquad \qquad p_1 = \begin{bmatrix} -1 \\ 0 \end{bmatrix}, \ p_2 = \begin{bmatrix} 0 \\ 0 \end{bmatrix}, \ p_3 = \begin{bmatrix} 1 \\ 0 \end{bmatrix}, \end{aligned}$$

## Linear approximations around an equilibrium

### Review: linearization around an equilibrium

Let  $(\bar{x}, \bar{u})$  be an equilibrium for the NL invariant system

 $\dot{x} = f(x, u)$ y = g(x, u)

Deviations:  $\delta x(t) = x(t) - \bar{x}$ ,  $\delta u(t) = u(t) - \bar{u}$ ,  $\delta y(t) = y(t) - \bar{y}$ 

First order Taylor expansion about the equilibrium:

$$f(x,u) \simeq f(\bar{x},\bar{u}) + D_x f(x,u) \Big|_{\substack{x=\bar{x}\\u=\bar{u}}} (x-\bar{x}) + D_u f(x,u) \Big|_{\substack{x=\bar{x}\\u=\bar{u}}} (u-\bar{u})$$
$$g(x,u) \simeq g(\bar{x},\bar{u}) + D_x g(x,u) \Big|_{\substack{x=\bar{x}\\u=\bar{u}}} (x-\bar{x}) + D_u g(x,u) \Big|_{\substack{x=\bar{x}\\u=\bar{u}}} (u-\bar{u})$$

$$D_{x}f(x,u) = \begin{bmatrix} \frac{\partial f_{1}(x,u)}{\partial x_{1}} & \cdots & \frac{\partial f_{1}(x,u)}{\partial x_{n}} \\ \vdots & \ddots & \vdots \\ \frac{\partial f_{n}(x,u)}{\partial x_{1}} & \cdots & \frac{\partial f_{n}(x,u)}{\partial x_{n}} \end{bmatrix}$$

Jacobian with respect to the variables x

### Review: linearization around an equilibrium

One gets:

$$\dot{\delta x} = \dot{x} - \dot{\bar{x}} = f(x, u) \simeq \underbrace{f(\bar{x}, \bar{u})}_{=0} + D_x f(x, u) \Big|_{\substack{x = \bar{x} \\ u = \bar{u}}} \delta x + D_u f(x, u) \Big|_{\substack{x = \bar{x} \\ u = \bar{u}}} \delta u$$
$$\delta y = -\bar{y} + y \simeq \underbrace{-g(\bar{x}, \bar{u}) + g(\bar{x}, \bar{u})}_{=0} + D_x g(x, u) \Big|_{\substack{x = \bar{x} \\ u = \bar{u}}} \delta x + D_u g(x, u) \Big|_{\substack{x = \bar{x} \\ u = \bar{u}}} \delta u$$

### Linearized system

Defining

$$A = D_x f(x, u) \Big|_{\substack{x = \bar{x} \\ u = \bar{u}}}, \ B = D_u f(x, u) \Big|_{\substack{x = \bar{x} \\ u = \bar{u}}}, \ C = D_x g(x, u) \Big|_{\substack{x = \bar{x} \\ u = \bar{u}}}, \ D = D_u g(x, u) \Big|_{\substack{x = \bar{x} \\ u = \bar{u}}}$$

the linearized system around the equilibrium  $(\bar{x}, \bar{u})$  is

$$\dot{\delta x} = A\delta x + B\delta u$$
  
 $\delta y = C\delta x + D\delta u$ 

### Review: linearization around an equilibrium

We hope state trajectories of the linearized system are good approximations of  $x(t) - \bar{x} \dots$  but this does not always happen

Example: (a): 
$$\dot{x} = x^3$$
, (b):  $\dot{x} = -x^3$ 

Linearized systems around  $\bar{x} = 0$  are the same:  $\delta x = 0 \Rightarrow \delta x(t) = x_0$  but NL systems have different behaviors



### Phase plane

For second-order systems it is possible to graphically study the projection of trajectories in the plane  $(x_1, x_2)$  that is called phase plane

### Example





### Some types of equilibria for second-order LTI systems Autonomous LTI system

$$\dot{x} = Ax$$

- The origin  $\bar{x} = 0$  is always an equilibrium state
- When x(t) ∈ ℝ<sup>2</sup> one can classify the behavior of state trajectories using the eigenvalues λ<sub>1</sub>, λ<sub>2</sub> of the matrix A.



### Some types of equilibria for second-order LTI systems Autonomous LTI system

$$\dot{x} = Ax$$

- The origin  $\bar{x} = 0$  is always an equilibrium state
- When x(t) ∈ ℝ<sup>2</sup> one can classify the behavior of state trajectories using the eigenvalues λ<sub>1</sub>, λ<sub>2</sub> of the matrix A.



### Equilibria of second-order NL systems

**Idea**: analyze the behavior of state trajectories around an equilibrium state using the linearized system

Example: Duffing model for  $\eta = 1$ Linearized system NL system  $\delta x_1 = \delta x_2$  $\dot{x}_1 = x_2$  $\dot{\delta x_2} = \delta x_1 - 3\bar{x}_1^2 \delta x_1 - \delta x_2$  $\dot{x}_2 = x_1 - x_1^3 - nx_2, n = 1$ Around  $p_1 = \begin{bmatrix} -1 & 0 \end{bmatrix}^T$  and  $p_3 = \begin{bmatrix} 1 & 0 \end{bmatrix}^T$  $D_{x}f = \begin{bmatrix} 0 & 1 \\ -2 & -1 \end{bmatrix} \Rightarrow$  Eigenvalues:  $-\frac{1}{2} \pm j\frac{\sqrt{3}}{2}$ Can we conclude that  $p_1$  and  $p_3$  are stable foci?

### Equilibria of second-order NL systems

Around  $p_2 = \begin{bmatrix} 0 & 0 \end{bmatrix}^{\mathrm{T}}$  $D_x f = \begin{bmatrix} 0 & 1 \\ 1 & -1 \end{bmatrix} \Rightarrow \text{ Eigenvalues:} -1 \pm \frac{\sqrt{5}}{2}$ 

Can we conclude that  $p_2$  is a saddle ?



From the state trajectories it seems the answer is yes... The analysis is local.

### Equilibria of second-order NL systems

Duffing model: global behavior



Plots obtained with the MatLab program *pplane* http://math.rice.edu/~dfield/

## **Subharmonics**

### Subharmonics



Duffing model with input  $\dot{x}_1 = x_2$   $\dot{x}_2 = x_1 - x_1^3 - \eta x_2 + u$  $\eta = 0.025, \ u(t) = 7.5 \sin(t)$ 



### Subharmonics

Harmonics that are NOT present in the input appear in the output (even in the asymptotic régime)

• Impossible for asymptotically stable LTI systems (because of the frequency response theorem)

## Chaos

### Chaos



Duffing model with input  $\dot{x}_1 = x_2$   $\dot{x}_2 = x_1 - x_1^3 - \eta x_2 + u$  $\eta = 0.025, \ u(t) = 7.5 \sin(t)$ 

State trajectory  $x_1$  when x(0) = 0 and  $x(0) = 0 + \epsilon$ 



### Chaos

- Huge sensitivity to initial states.
- Simulations might be meaningless

## Limit cycles

## Surge and rotating stall in jet engine compressors Engine "de Havilland Goblin II" $^{\rm 1}$



#### <sup>1</sup>Picture from Wikipedia

### Surge and rotating stall in jet engine compressors



### NL model (dimensionless units)

- B > 0 compressor angular speed (rotor)
- x<sub>1</sub>: compressor mass flow
- x<sub>2</sub>: plenum pressure rise
- $\alpha$ : throttle angle
- $C(\cdot)$ : compressor charachteristic
- $F_{\alpha}(\cdot)$ : throttle charachteristic

$$\dot{x}_1 = B(C(x_1) - x_2)$$
$$\dot{x}_2 = \frac{1}{B}(x_1 - F_{\alpha}^{-1}(x_2))$$

### Analysis of equilibria

### Computation of equilibria

$$\begin{cases} 0 = B\left(C(\bar{x}_1) - \bar{x}_2\right) \\ 0 = \frac{1}{B}\left(\bar{x}_1 - F_{\alpha}^{-1}(\bar{x}_2)\right) \end{cases} \Rightarrow \begin{cases} \bar{x}_2 = C(\bar{x}_1) \\ \bar{x}_1 = F_{\alpha}^{-1}(\bar{x}_2) \end{cases} \Rightarrow F_{\alpha}(\bar{x}_1) = C(\bar{x}_1) \end{cases}$$

### Equilibria for various throttle angle



### Unstalled operating point



This is the desired behavior: stable equilibrium

Ferrari <sup>1</sup>	Trecate (	(DIS)
		(2.0)

### Surge



Perturbation of the throttle charachteristic

- Unstable equilibrium and stable limit cycle
- Surge  $\Rightarrow$  dangerous pressure waves

### Rotating stall



Perturbation of the throttle charachteristic + decrease of the angular speed B of the compressor

• Stable equilibrium but insufficient pressure  $\Rightarrow$  rotating stall !